

SAE Baja - Drivetrain

Project Proposal

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Overview

- Introduction
- Needs and Constraints
- QFD/HOQ
- Problem Definition and Project Goal
- Transmission Choices
- Decision Matrix
- Design Analyses and Calculations
- Gantt Chart
- Conclusion

Introduction

- Choosing which transmissions to begin analyzing
- Continued research and analysis to further the understanding of the designs
- Gantt Chart

Needs and Constraints

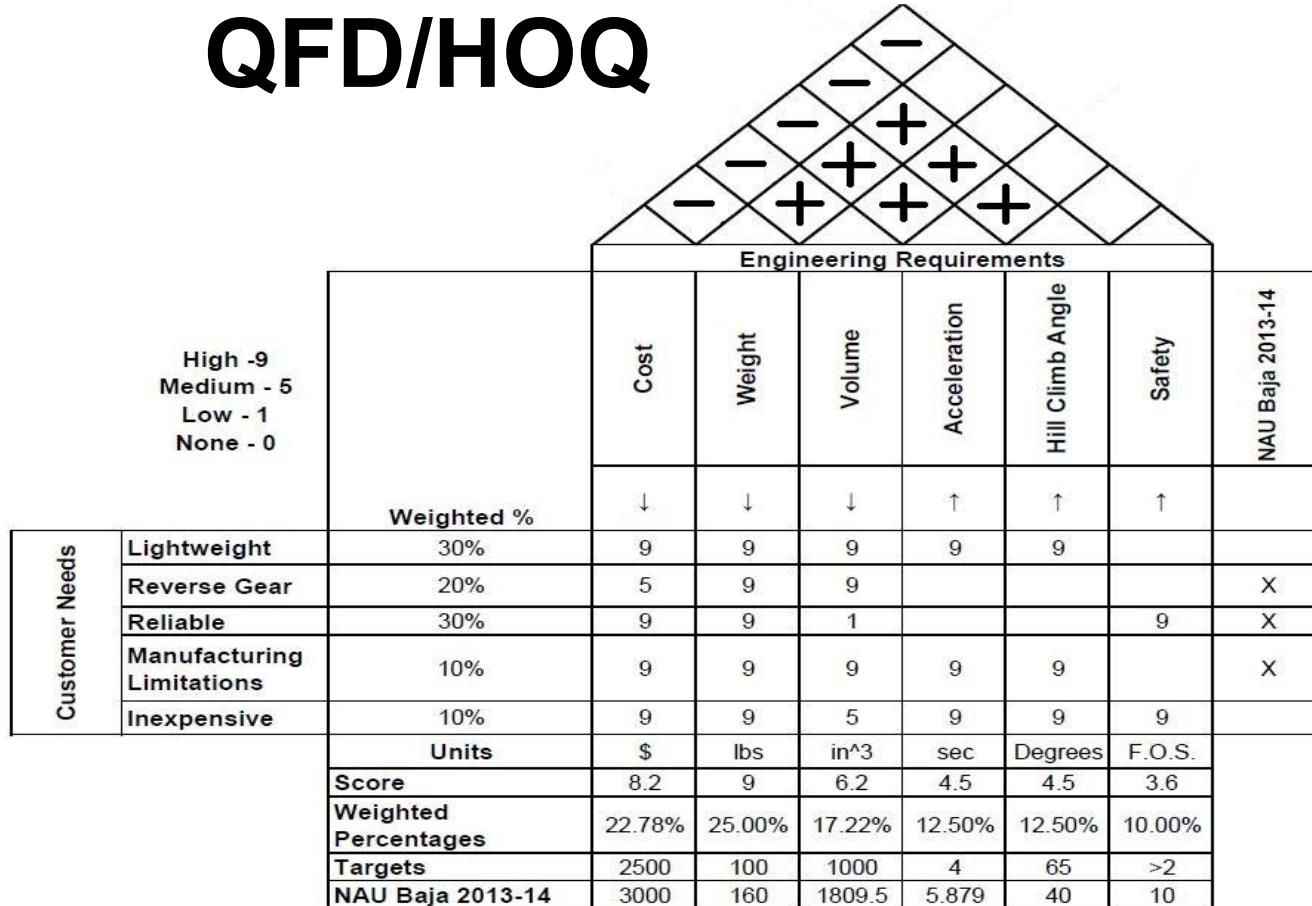
Dr. John Tester, Client

- Lightweight - vehicle is required to be, at most, 450 pounds in weight

Constraints

- Must use provided engine - Briggs & Stratton 10 horsepower OHV Intek
- Design drivetrain within SAE Baja rules
- Complete a 100 feet trial in 4 seconds on level, dry pavement
- Able to climb an incline of at least 60 degrees
- Possible to manufacture in NAU Machine Shop

QFD/HOQ



Problem Definition and Project Goal

Design and develop a drivetrain that is able to attain the desired torque and speed for the SAE Mini Baja in order to place in the top 10 in the Hill Climb and Acceleration challenges against competing universities.

Transmission Concepts

- Automatic
- CVT Belt
- CVT Gear
- Direct Drive
- Manual
- Sequential

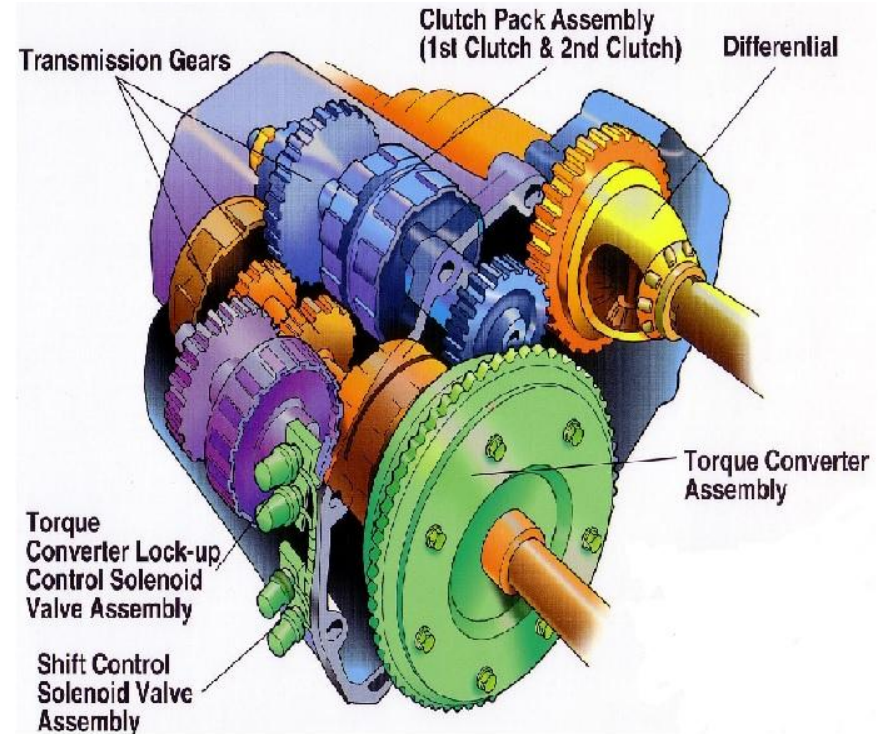
Automatic Transmission

Pros:

- High gear ratio range
- Reliable
- Reverse gear capable

Cons:

- Cost
- Medium efficiency
- Size
- Extremely difficult to design



(Automatic Transmission)

Direct Drive Transmission

Pros:

- Cost
- Simplicity of design
- Size
- Weight
- Highly efficient

Cons:

- Static gear ratio
- Not reverse gear capable



(Direct Drive)

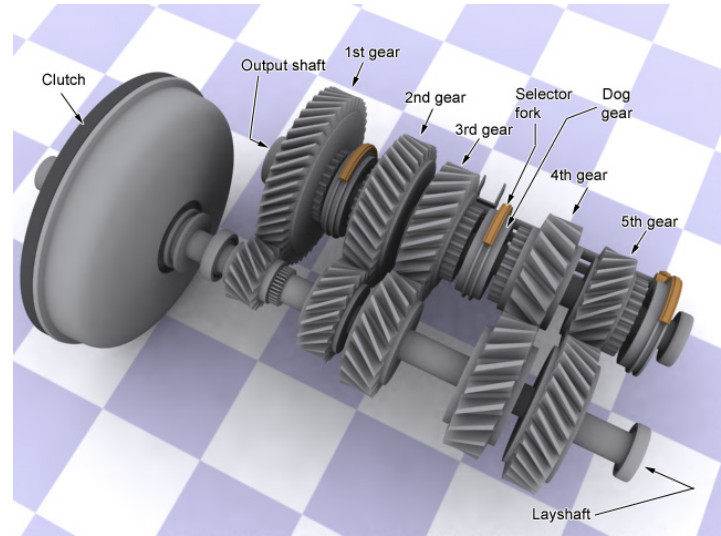
Manual Transmission

Pros:

- Reverse capable
- Reliable
- Cost effective

Cons:

- Extra weight from clutch
- Loss of power between shifts



(Representation of Manual Gearbox)

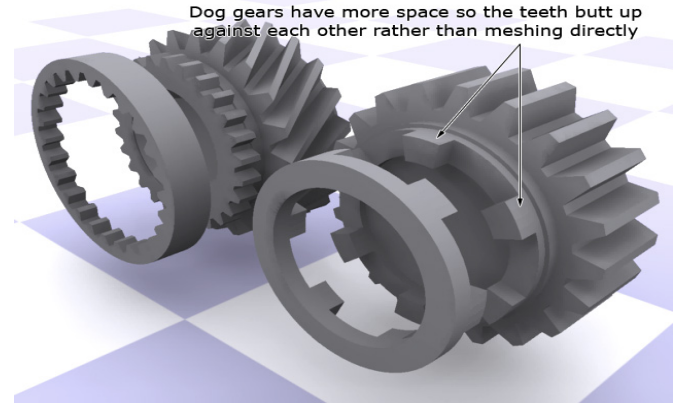
Sequential Transmission

Pros:

- Little loss of power
- Lightweight and compact
- Simple to operate
- Stronger and more reliable

Cons:

- Difficult to integrate reverse and clutch
- Possible increased cost



Dog gears have more space so the teeth butt up against each other rather than meshing directly

(Sequential Dog ring compared to Manual Dog ring)

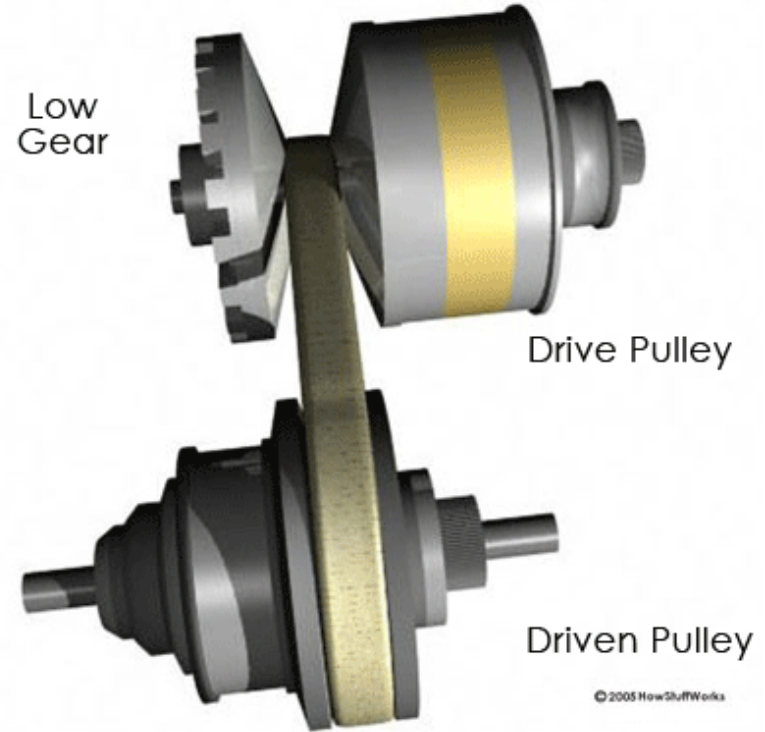
Continuous Variable Transmission (Belt)

Pros:

- Ease of use
- Size
- Weight

Cons:

- Cost
- Efficiency
- Reliability
- Reverse gear



(CVT Picture)

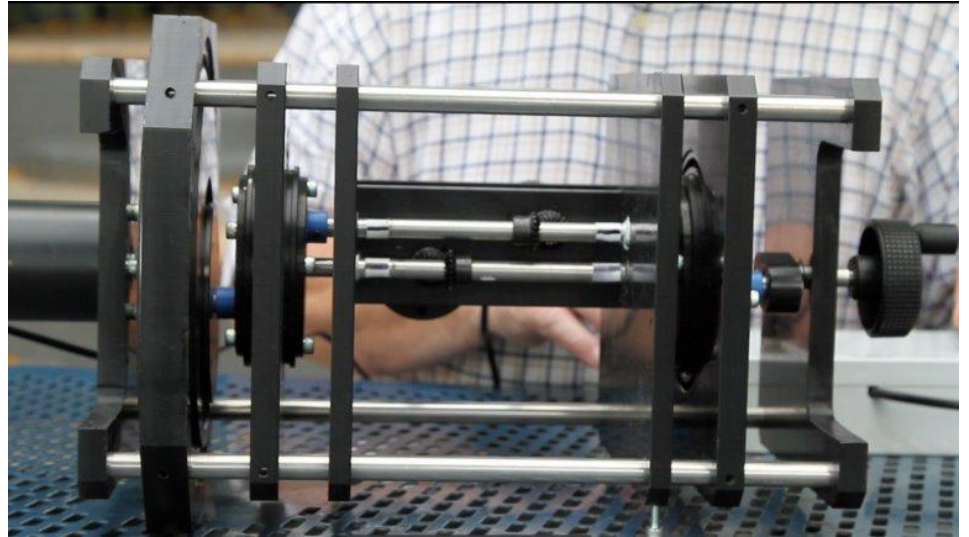
Continuous Variable Transmission (Gears)

Pros:

- Ease of use
- Variability of gear ratios
- Efficiency
- Reverse gear

Cons:

- Cost
- Weight
- Simplicity of design



(Gear CVT)

Original Decision Matrix

Scale 1-5 5 = Best, 1 = Worst	Cost	Gear Ratio Range	Efficiency (Loss of Power)	Weight	Simplicity of Design	Reliability	Size/Volume	Reverse Gear Capable	Total
Sequential	3	5	5	4	3	4	4	3	3.95
Manual	3	5	4	3	4	4	3	4	3.85
CVT Belt	2	3	2	3	5	2	5	1	2.35
CVT Gear	2	5	4	3	3	4	3	5	3.85
Automatic	2	4	3	3	2	4	2	4	3.2
Straight (One Gear Ratio)	5	2	5	5	5	5	5	1	3.75
Customer Weighting	15%	15%	20%	10%	5%	10%	5%	20%	

Manual versus Sequential

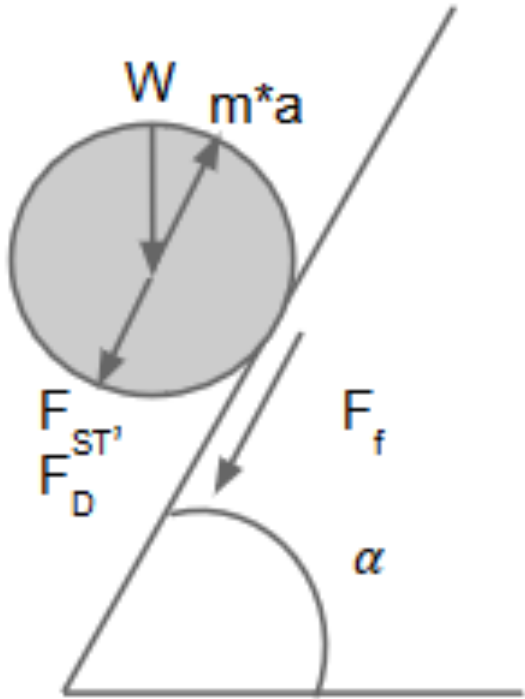
Manual:

- Reverse gear capable
- Simple design

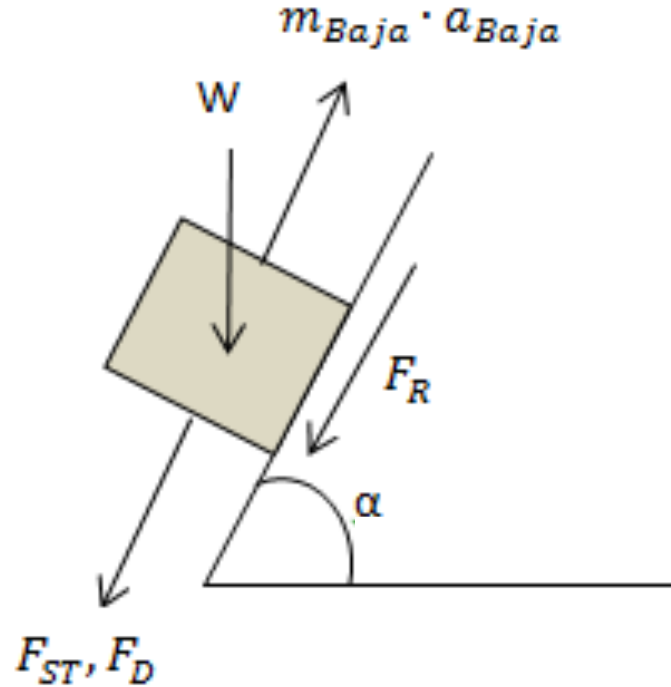
Sequential:

- Efficient
- High gear ratio range
- Fast gear selection

Hill Climb Free Body Diagrams



FBD of Wheel



FBD of Vehicle

Calculations - Hill Climb, Vehicle

- Givens/Assumptions:

- $W = 600 \text{ lb}$
- $f_R = 0.16$
- $c_W = 0.62$
- $P_{Z,B} = 8.5 \text{ hp} = 4675 \text{ lb}\cdot\text{ft} / \text{s}$
- $\alpha_{St} = 60^\circ$
- $A = 9.98 \text{ ft}^2$
- $\rho_{air} = 0.00228 \text{ slug}/\text{ft}^3$
- $v = 5 \text{ mph} = 22/3 \text{ ft/s}$

- Equations Used:

$$F_{St} = m_F g \sin \alpha_{St}$$

$$F_L = \frac{1}{2} \rho_L c_W A v^2$$

$$F_R = f_R m_F g \cos \alpha_{St}$$

$$F_{Z,B} = F_R + F_{St} + F_L + F_a$$

$$P_{Z,B} = F_{Z,B} v$$

Results - Hill Climb, Vehicle

- Results
 - $F_{St} = 519.615 \text{ lbf}$
 - $F_f = 48 \text{ lbf}$
 - $F_L = 0.379 \text{ lbf}$
 - $F_{Z,B} = 567.994 \text{ lbf}$
 - $v_{\text{vehicle}} = 5.616 \text{ mph} = 8.236 \text{ ft/s}$

Calculations - Hill Climb, Wheel

- First Gear Ratio Assumptions:

- Assume $F_{\text{total}} = 600 \text{ lb}$
- Assume $v_{\text{vehicle}} = 6 \text{ mph}$
- $P = 8.5 \text{ hp}$, $\alpha = 60^\circ$
- 22 in diameter tire, $R = 11 \text{ in}$
- $N_{\text{min}} = 1800 \text{ rpm}$
- $N_{\text{max}} = 2800 \text{ rpm}$

- Equations Used:

$$\omega = \frac{v}{r}$$

$$R = \frac{\omega_A}{\omega_B} = \frac{N_B}{N_A}$$

$$\tau = F \times r$$

Results - Hill Climb, Wheel

- Results
 - $\omega = 91.67 \text{ rpm}$
 - Gear Ratio_{min} @1800rpm = 19.63:1
 - Gear Ratio_{max} @2800rpm = 30.54:1
 - **Gear Ratio_{avg} = 24.1:1**
 - $T_{\text{wheel}} = 550 \text{ lb*ft}$

Calculations - Acceleration

- Assumptions
 - Distance = 100 ft
 - Time = 4 seconds
 - $a = 12.5 \text{ ft/s}^2$
 - $v = 23 \text{ mph}$
 - $m = 18.65 \text{ lbm}$
 - $R = 11 \text{ in}$
 - $A = 9.92 \text{ ft}^2$
 - $c_W = 0.62$
 - $\rho_L = 0.00228 \text{ slug/ft}^3$
 - $f_R = 0.014$

- Equations Used:

$$x = x_0 + v_0 t + \frac{1}{2} a_c t^2$$

$$F = ma$$

$$F_R = f_R m_F g \cos \alpha_{St}$$

$$F_L = \frac{1}{2} \rho_L c_W A v^2$$

Results - Acceleration

Results:

- $F_{\text{total, High}} = 250 \text{ lbf}$
- $F_{\text{total, Low}} = 241 \text{ lbf}$

$$\text{High Ratio} - \frac{F_{\text{Total,High}}}{2} \times \frac{13R}{12}$$

- High Ratio = **6.2 : 1**
- Low Ratio = **11.1 : 1**
- Total Time = 4.25 seconds

$$\text{Low Ratio} - \frac{F_{\text{Total,Low}}}{2} \times \frac{10R}{12}$$

Gear Ratios

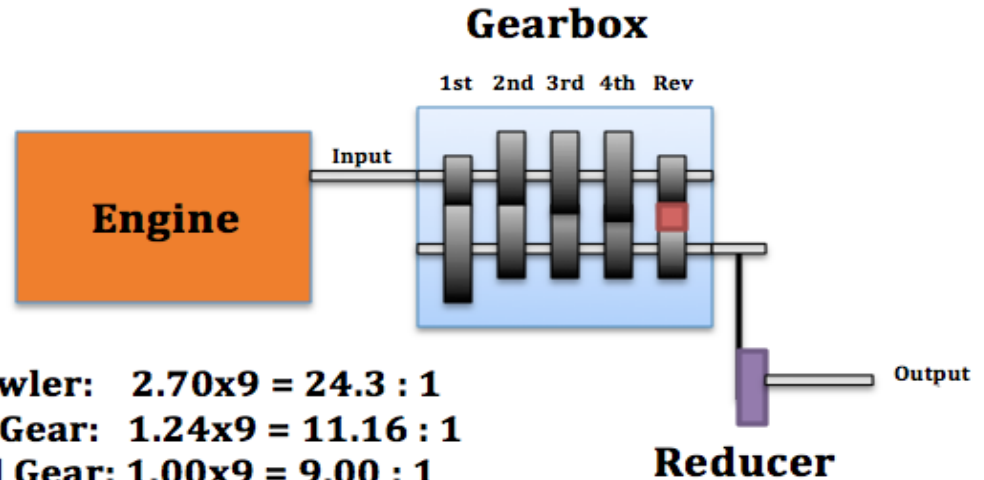
Engine to Gearbox Ratio: 1 : 1

Gear Box Ratios

- Crawler: **2.70 : 1**
- 1st: **1.24 : 1**
- 2nd: **1 : 1**
- 3rd: **0.696 : 1**
- Reverse: **1.20 : 1**

Reducer Ratio: 9 : 1

Crawler: $2.70 \times 9 = 24.3 : 1$
1st Gear: $1.24 \times 9 = 11.16 : 1$
2nd Gear: $1.00 \times 9 = 9.00 : 1$
3rd Gear: $.696 \times 9 = 6.26 : 1$
Reverse: $1.20 \times 9 = 10.8 : 1$



Gear Specification

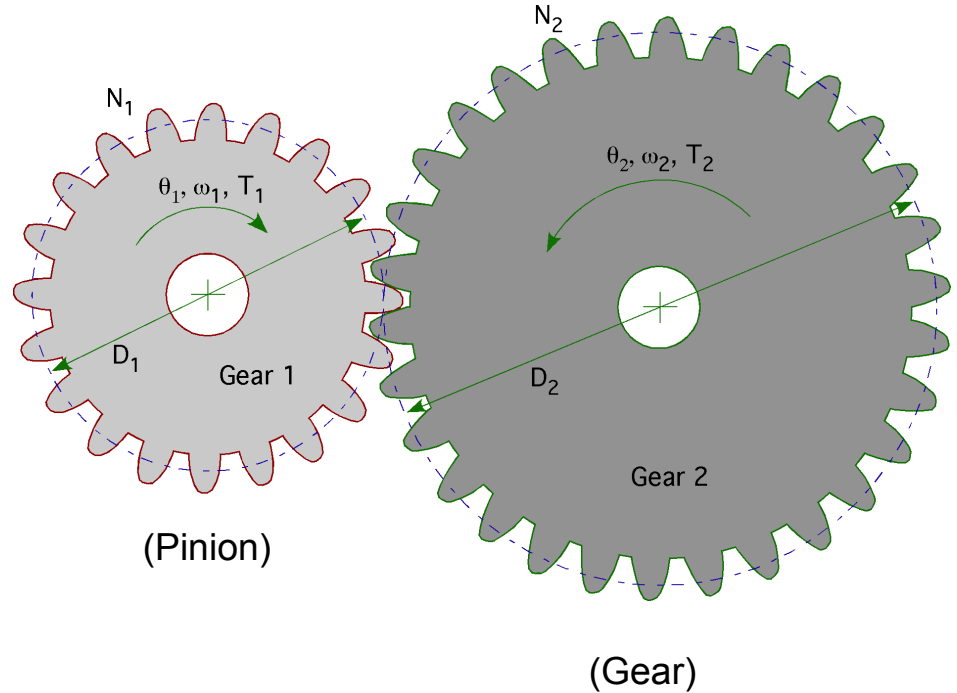
Gearbox

Crawler -	Pinion: 15	Gear: 41
1 st Gear -	Pinion: 25	Gear: 31
2 nd Gear -	Pinion: 28	Gear: 28
3 rd Gear -	Pinion: 33	Gear: 23
Reverse -	Pinion: 15	Gear: 15
		Gear 2: 18

Reducer - **Pinion:** 10 **Gear:** 90

Gear Material: 7075-T6 Aluminum

Minimum Factor of Safety: 6.2



Shafts

Input shaft $M_a = 578$ lb-in

Output shaft $M_a = 578$ lb-in

Input shaft $T_m = 192$ lb-in

Output shaft $T_m = 528$ lb-in

DE-Goodman:

$$\frac{1}{n} = \frac{16}{\pi d^3} \left\{ \frac{1}{S_e} [4(K_f M_a)^2 + 3(K_{fs} T_a)^2]^{1/2} + \frac{1}{S_{ut}} [4(K_f M_m)^2 + 3(K_{fs} T_m)^2]^{1/2} \right\}$$

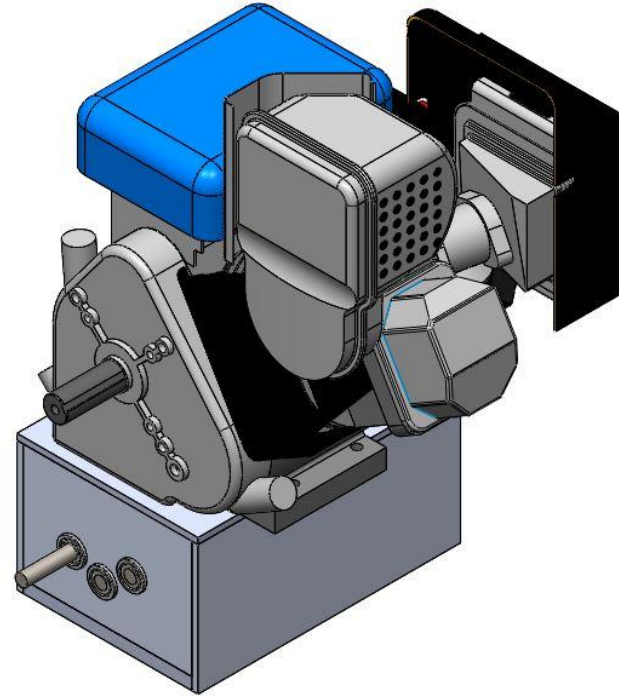
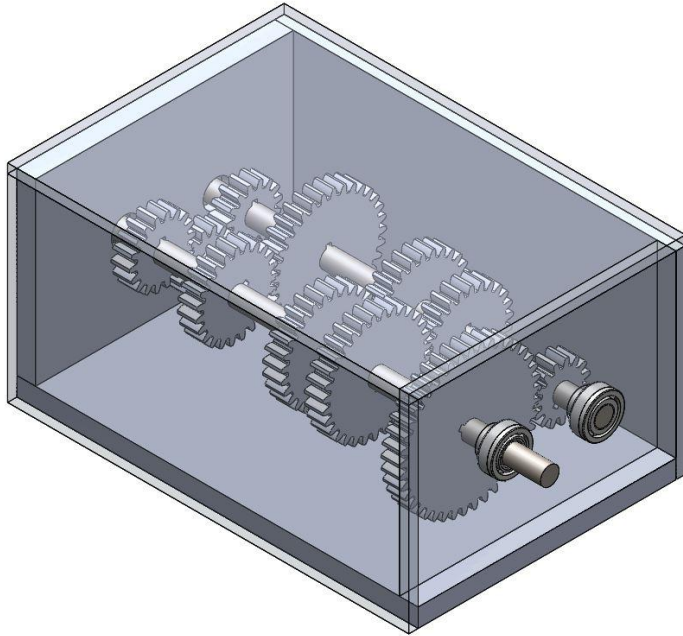
- Shaft Material: 4340 Normalized Steel
- 0.5 inch diameter solid shaft
- Factors of Safety: Input shaft = 2.94
 Output shaft = 2.00

Bearings

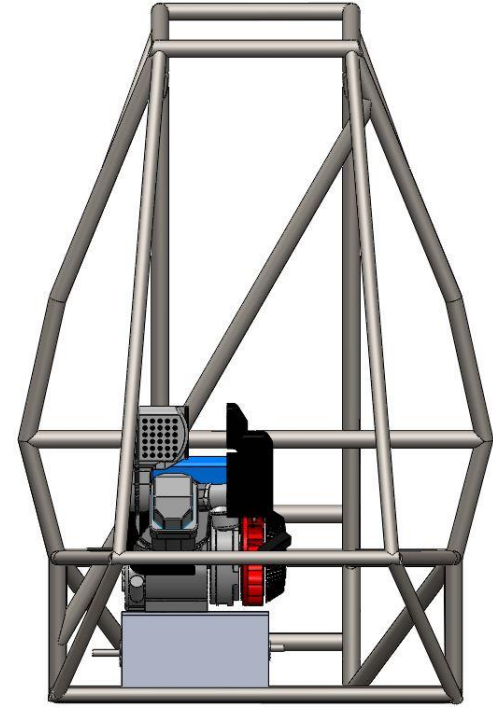
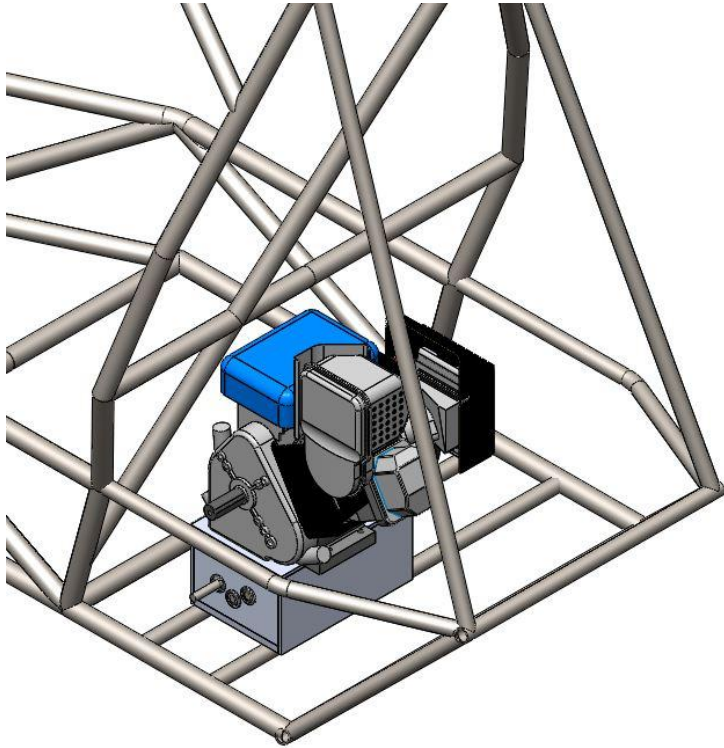
- McMaster-Carr Open Steel Ball Bearing
 - Shaft Diameter = 0.5 inches
 - Outside Diameter = 1.125 inches
 - Width = 0.375 inches
 - Dynamic Load Capacity = 600 pounds
- Factor of Safety: 2.3



CAD Drawing



Overall Assembly



Bill of Materials

Materials	Quantity	Cost for One Unit of Material	Overall Cost of Each Material	Free/Donated
7075 T6 Aluminum (4" diameter, 2' bars)	1	\$307.44	\$307.44	x
7075 T6 Aluminum (3" diameter, 5' bars)	1	\$298.87	\$298.87	x
7075 T6 Aluminum (2" diameter, 4' bars)	1	\$87.24	\$87.24	x
6061 T6 Aluminum (0.5" thick, 1'x3' plates)	1	\$164.92	\$164.92	x
6061 T6 Aluminum (0.25" thick, 1'x3' plates)	1	\$76.69	\$76.69	x
4340 Normalized Steel (5/8" inch diameter, 5' bar)	2	\$95.64	\$191.28	x
Bearings	6	\$7.36	\$44.16	
Clutch	1	\$300.00	\$300.00	
Differential	1	\$400.00	\$400.00	x
80 tooth sprocket	1	\$25.00	\$25.00	
10 tooth sprocket	1	\$10.00	\$10.00	

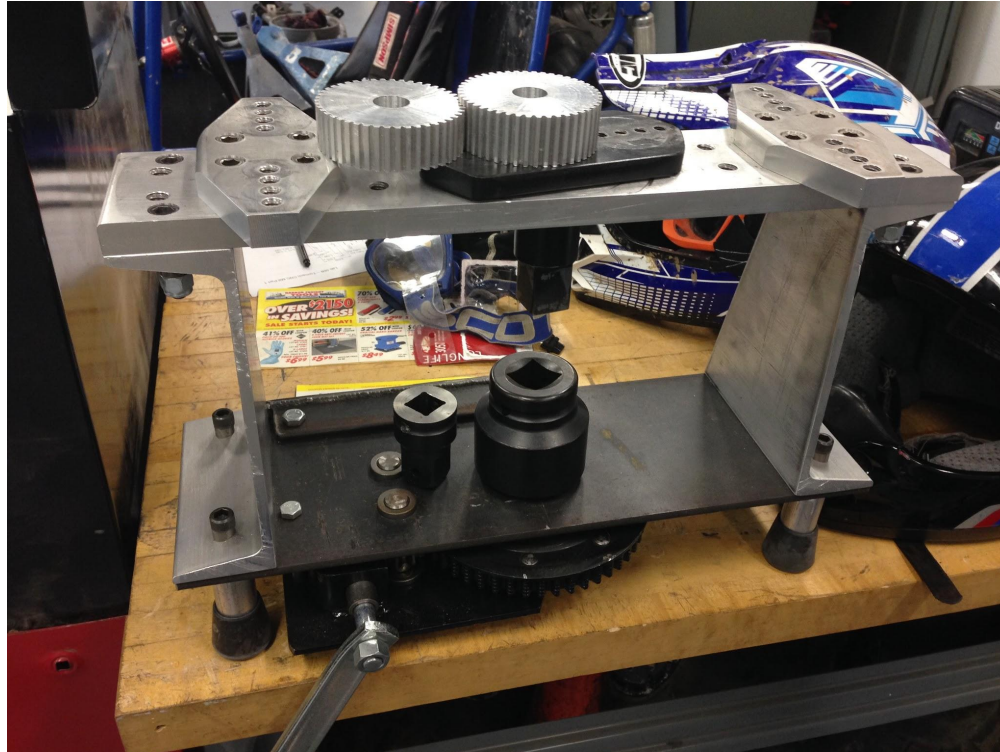
Total \$1,905.60

Total, subtracting free/donated \$379.16

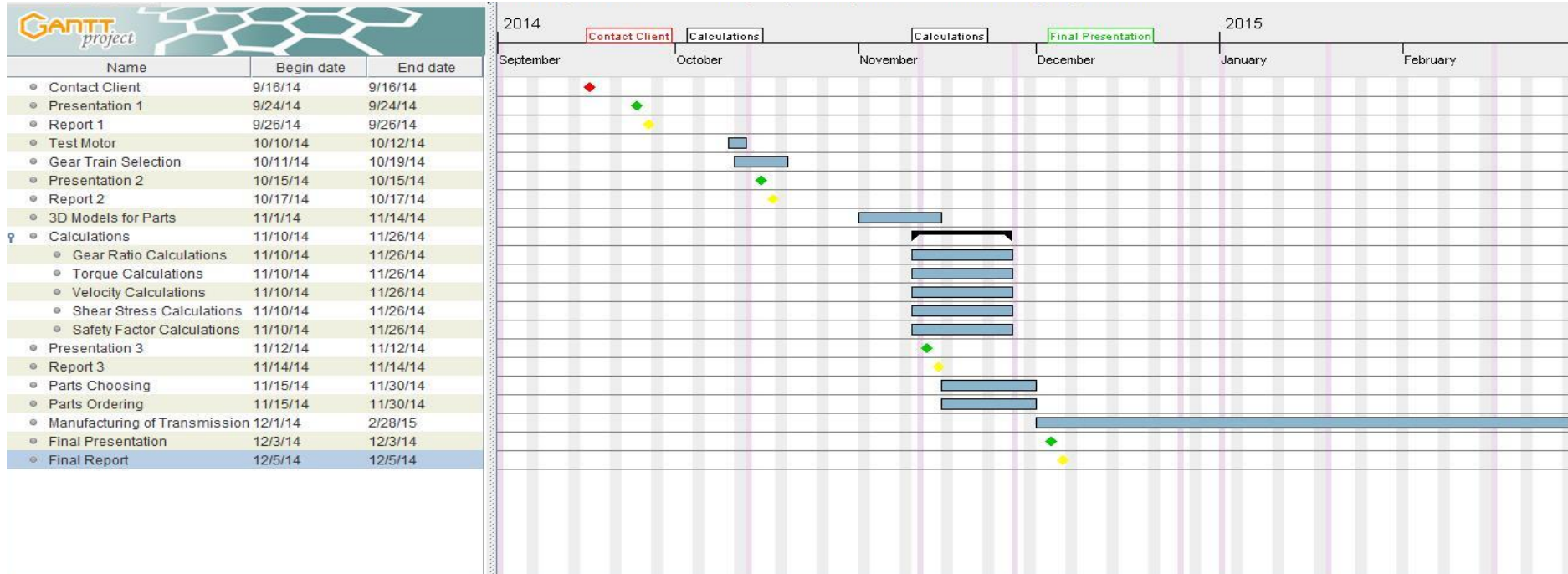
Manufacturing of Gearbox

- Can manufacture gears on site at the NAU Machine Shop
- Will either order or turn down metal received from Industrial Metal Supply Co.
- Purchasing bearings from McMaster-Carr and a clutch online

Manufacturing of Gearbox



Gantt Chart



Conclusion

- Selected, designed, and analyzed a sequential transmission
- Will begin to manufacture the gears during the month of December
- Will order the metal for the shafts and housing through Industrial Metal Supply Co. (IMS)
- Will purchase bearings and a clutch

References

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- Gear CVT Picture <http://www.gizmag.com/steve-durnin-d-drive-d-drive-infinitely-variable-transmission-geared/15088/picture/114606/>
- Automatic Transmission Picture <http://hdabob.com/Transmission.htm>

Questions?