Engineering Analysis

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Overview

- Problem Statement
- Chassis Analysis
- Braking Analysis
- Steering Analysis
- Project Plan
- Conclusion

Problem Statement

• Design a vehicle that maximizes fuel efficiency for the Shell Eco-marathon competition.

Chassis Analysis

-Minimize vehicle frontal area while maintaining a comfortable driving position and adequate driver visibility.



-Fairing tail section reduction should not exceed 22 degrees in the YZ or XZ plane to ensure flow separation does not occur.

-Chassis floor should taper between 3-4 degrees towards the rear of the vehicle to reduce turbulence of the merging flow paths coming from above and below the vehicle.

Frontal Area/Seat Angle



Aerodynamic Drag



Jericho Alves

Chassis Rigidity

$$\delta_{max} = \frac{Fa(L^2 - a^2)^{3/2}}{9\sqrt{3}LEI}$$

$$x_1 = \sqrt{\frac{L^2 - a^2}{3}}$$

Variable		Value		
a (Load to nearest support)	.6 m			
L (Wheelbase)	2.5 m			
X (Point of maximum deflection)	1.484 m			
E (Elastic Modulus)		141 GPa		
I (Moment of Inertia)	.079 m^4			
Load at a	Maximum deflection at x			
60 kg	1.19 mm			
90 kg	1.78	'8 mm		
120 kg	2.37 mm			

Braking Analysis

 Each braking system must hold car at 20% grade



Braking Analysis

- Most mountain bike braking systems can provide enough force.
- Brake pads range in material, cost, strength.
- Rotor sizes 160mm, 185mm, and 203mm.



Steering Analysis

 Ackermann Steering Geometry

 $\cot \delta_o - \cot \delta_i = \frac{w}{l}$

- Track width (w)100-130cm
- Wheelbase (I) 220-350cm



$$R=\sqrt{a_2^2+l^2\cot^2\delta}$$

 $a^2 = 120cm$

l = 220cm - 350cm

Rolling Resistance

F = CrrN =.0025x1111.5 = 2.79N F - rolling resistance force Crr - coefficient of rolling friction N - normal force

Torque T= Fr = 2.79x.508=1.42 Nm

r - radius of the wheel



Project Plan

		G	Project	4	凶	2013			2014	
			Name	Begin date	End date	October	l November	l December	l January	l February
9 (0	Chassis	s Design	10/6/13	11/15/13					
		• Stee	ering System Design	10/6/13	10/20/13					
		• From	nt Subframe Design	10/21/13	10/31/13	i i				
		• Rea	r Subframe Design	10/21/13	10/31/13					
		Fairi	ing Design	11/1/13	11/15/13					
9	0	Chassis	s Construction	11/16/13	2/17/14		-			
		• Orde	ering Chassis/Fairi	11/16/13	12/16/13					
		• Cha	ssis/Monocoque C	12/17/13	2/17/14					No. of Concession, Name
		• Fairi	ing Construction	12/17/13	2/17/14					
		• Orde	ering Steering Syste	.11/16/13	12/16/13					
		• Stee	ering System Constr	.12/17/13	2/17/14					

Conclusion

- The overall size of the vehicle fairing will be determined by the desired seating angle between 15 and 30 degrees.
- Each braking system must hold car at a 20% grade slope.
- 160mm rotors and semi-metallic brake pads are ideal for low speeds and forces.
- Nearly all disc brake systems for mountain bikes are strong enough.
- In calculating the radius, the best results are track width of 123cm, wheelbase length of 320cm, and rolling resistance of 2.79Nm.

References

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- J. Walker, Jr., "The Physics of Braking Systems" (1st Ed.) [tab] http://www.stoptech.com/docs/media-centerdocuments/the-physics-of-braking-systems, 2005.
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Questions?