

# SAE Baja Design Final Design Presentation Team Drivetrain

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# Overview

- Introduction
- Concept Generation and Selection
- Engineering Analysis
- Cost Analysis
- Project plan
- Conclusions
- References

# Introduction

- Introduction of Competition
- Goal Statement
- Objectives
- Needs Identification
  - Customer Needs
  - Engineering Requirements
  - Quality Function Deployment (QFD)
- Product Specification
  - Requirements
  - Constrains

# Introduction of Competition

- Sponsored by SAE
- Project Description: Design and build Baja vehicle
- Participants: over 100 universities
- The engine is provided: common ground
- The Baja competition project in NAU:
  - Frame design
  - Suspension design
  - Drive-train design

# Goal Statement

- To build a rigid and durable Baja vehicle that can successfully complete all of the SAE competition events.

# Objectives

- Satisfy the client and stakeholder needs and requirements.
- Build a drive-train for the Baja vehicle so that it can complete the following tests successfully:
  - Acceleration
  - Traction
  - Maneuverability
  - Specialty
  - Endurance

# Customer Needs

- Most important customer needs:
  - Ability to climb the hill
  - Ability to pull an excess load
  - Able to reverse
  - Large max Velocity
  - Durability
  - Inexpensive

# Engineering requirements

- Customer Needs described by Engineering Requirements
  - Material strength (Kpa)
  - Torque (N-m)
  - Power efficiency (%)
  - Velocity (m/s)
  - Cost (\$)



# Quality Function Deployment (QFD)

| Customer Needs                      | Engineering Requirements for Drive-train |         |                |        |        |          |                   |                  |
|-------------------------------------|--|---------|----------------|--------|--------|----------|-------------------|------------------|
|                                     | Customer Weights                         | Cost    | Size           | Torque | Weight | Velocity | Material strength | Power efficiency |
| Safety                              | 7  | 3       |                |        | 3      | 1        | 9                 |                  |
| Accelerate fast                     | 8  | 1       | 3              | 9      | 3      | 3        |                   | 9                |
| Able to climb the hill              | 10                                       | 1       |                | 9      | 3      | 3        |                   | 3                |
| Able to pull an excess load         | 10                                       | 1       |                | 9      | 3      | 1        |                   | 3                |
| Durability                          | 9  | 3       |                |        |        |          | 9                 |                  |
| Long maintenance period             | 5  | 3       | 3              |        |        |          | 9                 |                  |
| Drive fast                          | 10                                       | 1       |                | 3      | 3      | 9        |                   | 3                |
| Able to reverse                     | 8  |         |                | 9      |        |          |                   |                  |
| Inexpensive                         | 7  | 9       | 1              |        | 1      |          | 3                 |                  |
| <b>Raw score</b>                    |  | 164     | 46             | 354    | 142    | 161      | 210               | 162              |
| <b>Relative Weight</b>              |  | 13%     | 4%             | 29%    | 11%    | 13%      | 17%               | 13%              |
| <b>Unit of Measure</b>              |  | Dollors | m <sup>3</sup> | N.m    | kg     | m/s      | Kpa               | ul               |
| <b>*ul--&gt; Unitless by method</b> |  |         |                |        |        |          |                   |                  |

# Product Specification

- 2014 Collegiate Design Series Baja SAE rules govern the requirements and constraints of our design.
- This was provided to us through SAE and explicitly states what is legal and illegal in the competition.

# Requirements

- All requirements were implied not stated.
- Select and Design a transmission given a specific motor that will allow you to complete multiple strenuous tasks.
- This transmission should be able to withstand repeated performances of each task.

# Constraints

- Because we are required to design for this part there were not too many constraints specifically for the drive-train.
- The Briggs & Stratton motors are governed at 3800 RPMs.

# Concept Generation and Selection

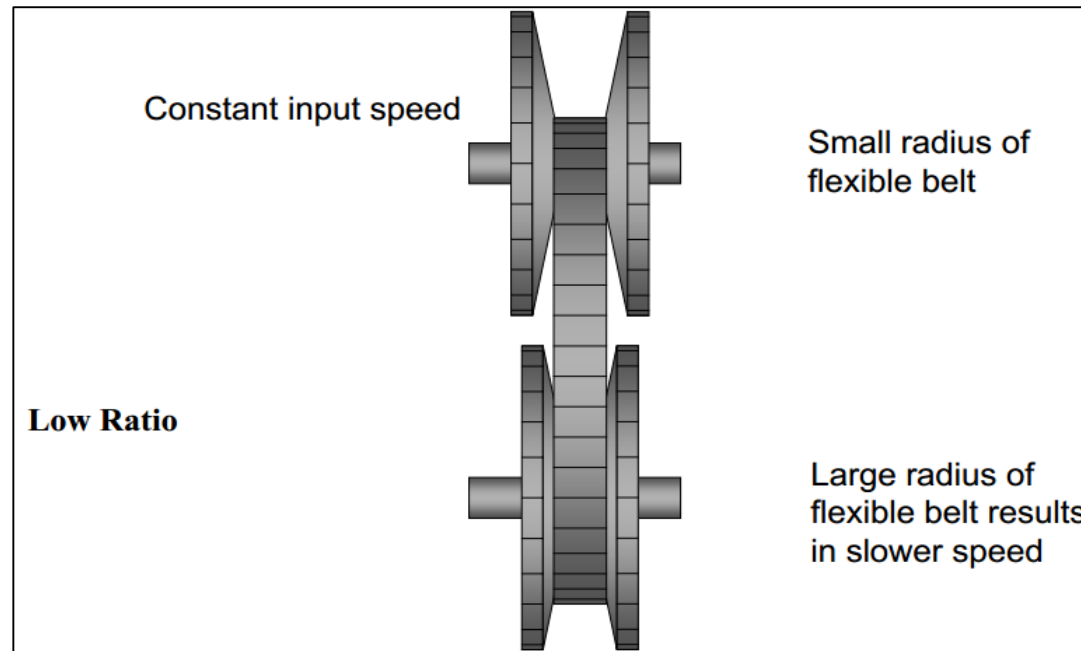
- Concepts Generation
  - Objective
  - Continuously Variable Transmission
  - Automatic Transmission
  - Manual Transmission
- Concept Selection
  - Decision Matrix

# Objective

- The purpose of our team is to design the best possible drivetrain for the specific use of a single seat off road Baja.
- Our most pressing issue is the identification of the best possible transmission.
  - CVT
  - Automatic
  - Manual

# Continuously Variable Transmission (CVT)

- Transfer the range of power and torque from engine continuously.
- The specific CVT is the Pulley drive



Source: cloudfront.net

# Continuously Variable Transmission (CVT)

## Advantages

- Do not need to shift gears
- Transfer the power continuously
- Good fuel efficiency
- Have a wide range of gear ratio

## Disadvantages

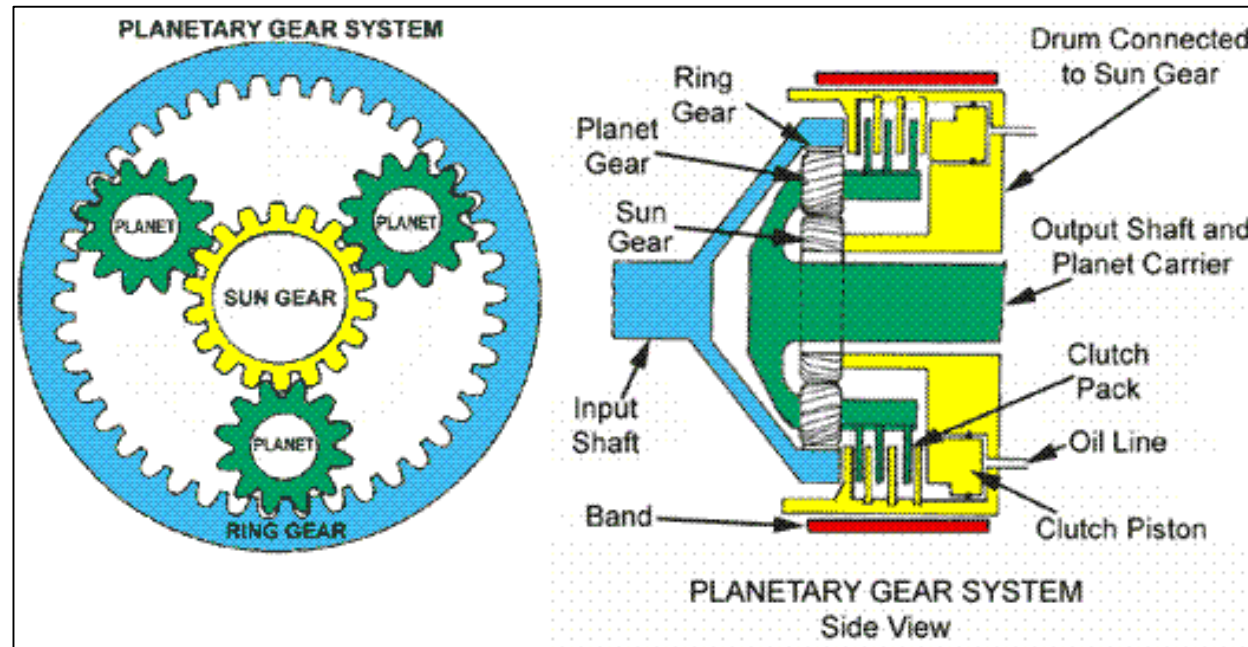
- The system cannot afford too much torque.
- Do not have a reverse.



# Automatic Transmission

- Can automatically change gear ratios as the vehicle cycles from low rpm to high rpm.

Planetary Gear System



Source: carparts.com

# Automatic Transmission

## Advantages

- Good performance in rough road.
- Easy to drive.
- Low failure rate.

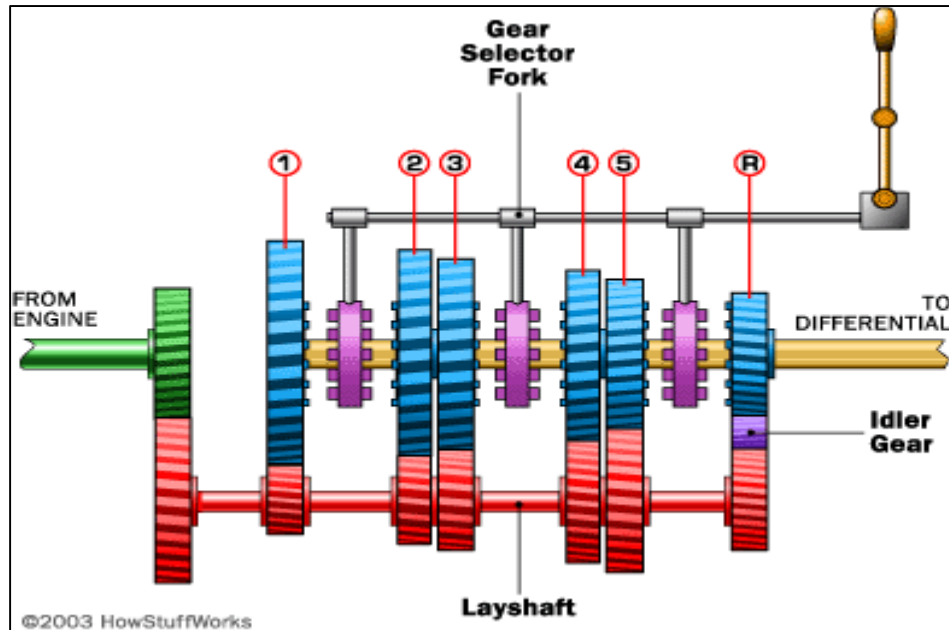
## Disadvantages

- Lower fuel efficiency.
- Higher price.
- Higher maintenance cost.

# Manual Transmission

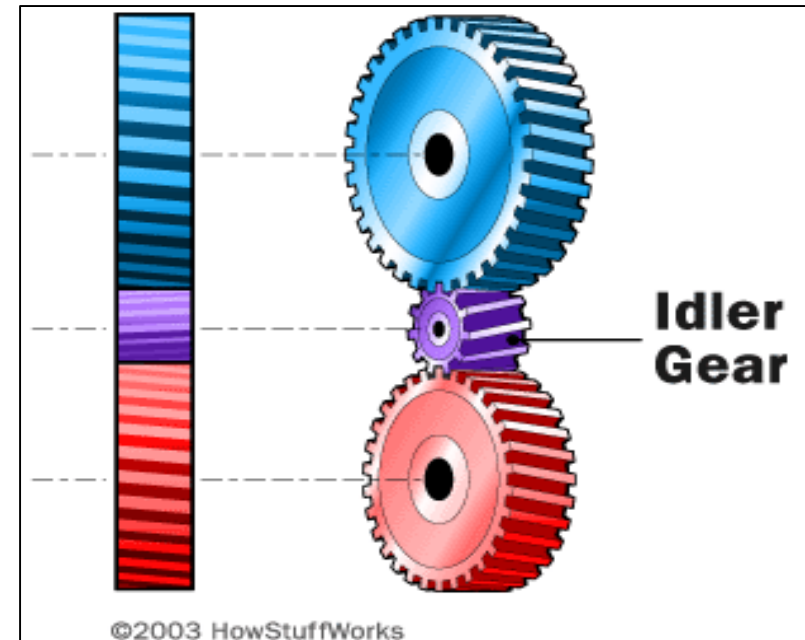
- Switch between the different gear ratios manually.

Manual Transmission Basic Concept



source: howstuffworks.com

Idler Gear



Source: howstuffworks.com

# Manual Transmission

## Advantages

- The driver has the ability to switch gears for higher rpm which helps in hill climbing.
- Allow for a better acceleration as the driver can switch gear to maximize torque.

## Disadvantages

- Low drivability.
- Low efficient comparing with CVT transmission.

# Concept Selection

Decision matrix

| <b>Concepts</b>   | <b>CVT</b> | <b>AT</b> | <b>MT</b> | <b>WEIGHT</b> |
|-------------------|------------|-----------|-----------|---------------|
| Durability        | 1          | 2         | 3         | 10%           |
| Maintenance       | 2          | 1         | 3         | 5%            |
| cost              | 1          | 2         | 3         | 15%           |
| Reversibility     | 2          | 3         | 1         | 10%           |
| Drivability       | 3          | 2         | 1         | 25%           |
| Acceleration      | 3          | 1         | 2         | 15%           |
| Energy Efficiency | 2          | 1         | 3         | 10%           |
| Weight            | 3          | 2         | 1         | 10%           |
| Weighted Total    | 2.25       | 1.8       | 1.95      | 100%          |

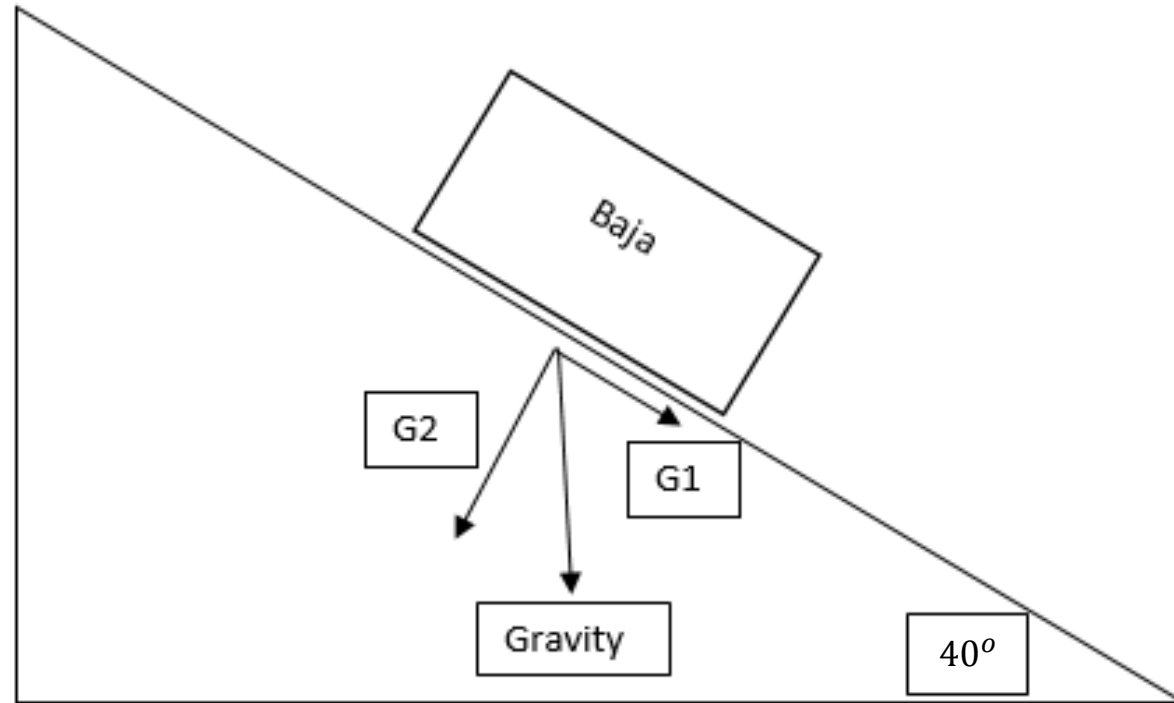
# Engineering Analysis

- Assumptions
- Goal
- Calculations
  - Ratio and Torque
  - Shaft, Key and Bearing
- Results
- Final Design

# Assumptions

- Wheel diameter( $D$ ): 23 inch
- Total weight ( $W$ ): 600 lb (including the driver)
- Slope of the hill ( $\theta$ ): 40 degree

# Goal (Hill Climb)





# Goal (Hill Climb)

- $G_1 = G \times \sin \theta = 600\text{lb} \times \sin 40 = 385.67 \text{ lb}$
- Force per wheel =  $192.83 \text{ lb}$
- Torque per wheel =  $192.83 \times \frac{D}{2} = 192.83 \times \frac{23}{2} \times \frac{1}{12} = 184.8 \text{ lb} - \text{ft}$
- Total torque ( $T_t$ ) =  $369.6 \text{ lb} - \text{ft}$

# Goal (Acceleration)

| Rank | Car No | School                              | Team                   | Time Run 1 | Time Run 2 | Best Time | Acceleration Score (75) |
|------|--------|-------------------------------------|------------------------|------------|------------|-----------|-------------------------|
| 1    | 1      | Cornell Univ                        | Big Red Racing         | 3.870      | 3.861      | 3.861     | 75.00                   |
| 2    | 52     | Michigan Tech Univ                  | Blizzard Baja          | 3.950      | 3.872      | 3.872     | 74.70                   |
| 3    | 6      | Univ of Maryland - Baltimore County | UMBC Racing            | 3.902      | 3.957      | 3.902     | 73.86                   |
| 4    | 78     | Univ of Maryland - College Park     | Terps Racing           | 3.906      | 3.974      | 3.906     | 73.75                   |
| 5    | 73     | LeTourneau Univ                     | Renegade Racing        | 3.935      | 3.916      | 3.916     | 73.48                   |
| 6    | 3      | Rochester Institute of Technology   | RIOT Racing            | 3.999      | 3.924      | 3.924     | 73.26                   |
| 7    | 44     | Ohio Northern Univ                  | Polar Bear Racing      | 3.945      | 3.955      | 3.945     | 72.67                   |
| 8    | 36     | Universite de Sherbrooke            | Sherbrooke Racing Team | 4.011      | 3.992      | 3.992     | 71.37                   |
| 9    | 57     | Univ of Wisconsin - Madison         | UW Baja                | 4.129      | 4.037      | 4.037     | 70.13                   |
| 10   | 45     | Univ of Arkansas - Fayetteville     | Racing Razorbacks      | 4.043      |            | 4.043     | 69.96                   |

Source: sae.org

# Goal (Acceleration)

- The top teams averaged 4 seconds to finish a 100 ft course.
- Assuming constant acceleration, we can calculate the maximum velocity:

$$\text{Distance} = \text{Max Velocity} \times \text{time} \div 2$$

$$\text{Max velocity} = \text{Distance} \times 2 \div \text{time}$$

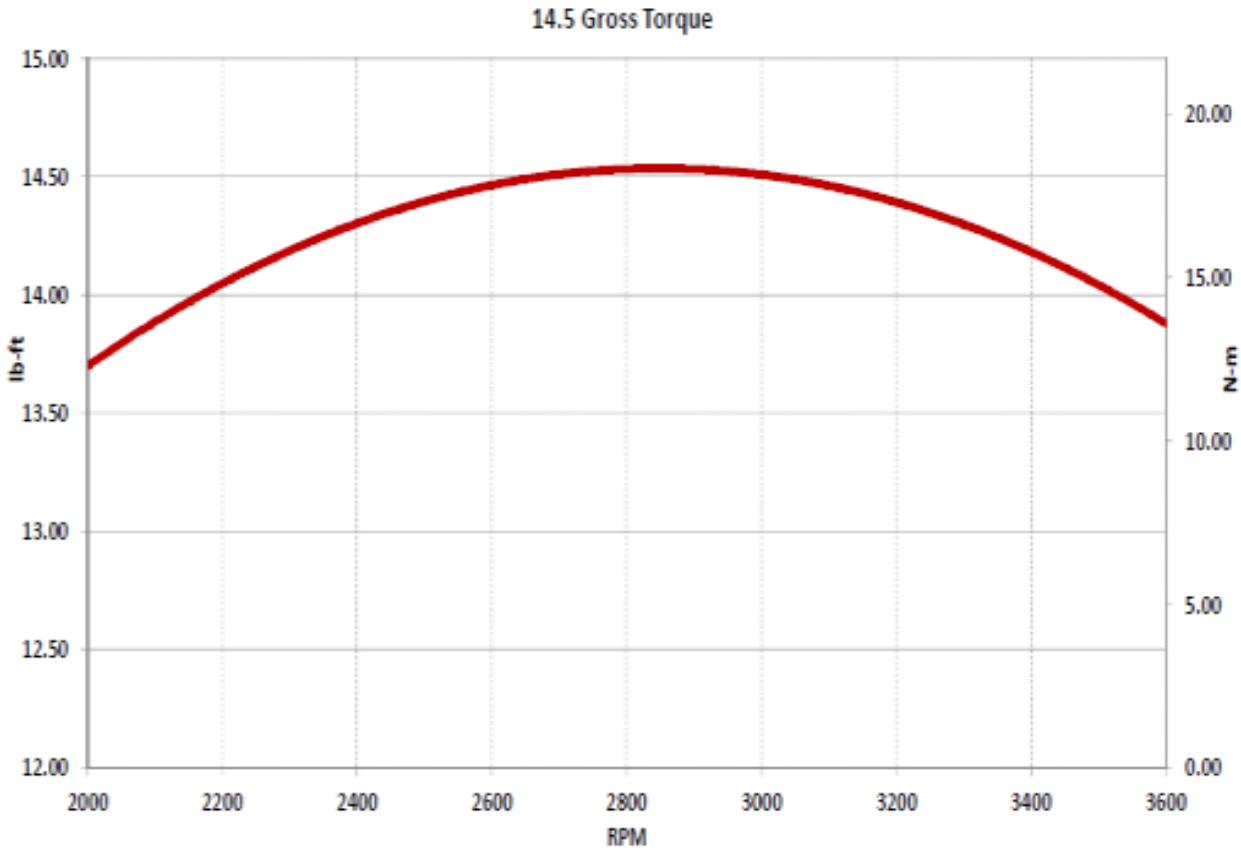
$$= 100 \times 2 \div 4$$

$$= 34 \text{ mph}$$

# Variables

- Chosen CVT: *PULLEY SERIES 0600 AND DRIVEN PULLEY SERIES 5600* from CVTech-AAB Inc.  
High speed ratio ( $r_{cvt-h}$ ) : 0.45 Low speed ratio ( $r_{cvt-l}$ ) : 3.1
- Chosen Gearbox: *ATV/UTV Gearbox T03* from GaoKin Inc.  
High speed ratio ( $r_{gb-h}$ ): 2.734 Low speed ratio ( $r_{gb-l}$ ): 5.682
- Second reduction ratio (Sprockets) ( $r_r$ ): 3
- Efficiency of CVT ( $N_{cvt}$ ): 88%
- Total ratio ( $r_t$ )
- Torque on the wheel (T)

# Torque Curve



Source: Briggs & Stratton

# Calculations (Ratio and Torque)

- Start RPM for CVT is 800 rpm and high speed ratio occur at 3600 rpm, assuming ratio varies linearly, we find the following relationship:

$$r_{cvt} = \begin{cases} 0 & \text{for } rpm < 800 \\ 3.1 - \frac{2.65 \times (rpm - 800)}{2800} & \text{for } 800 < rpm < 3600 \\ 0.45 & \text{for } 3600 < rpm \end{cases}$$

- Total ratio ( $r_t$ ) =  $r_{cvt} \times r_{gb} \times r_r \times N_{cvt}$

- T = Torque output from engine  $\times r_t$

- Speed =  $\frac{D \times RPM \times \pi}{total\ ratio \times 12 \times 60} \times 0.68 = \frac{23\ in \times RPM \times \pi}{total\ ratio \times 12 \times 60} \times 0.68$

# Calculations (Ratio and Torque)

Torque and Speed with high gearbox ratio

| Engine rpm | Torque output (lb-ft) | CVT ratio | Total ratio | Torque on wheel (lb-ft) | Speed (mph) |
|------------|-----------------------|-----------|-------------|-------------------------|-------------|
| 1800       | 13.20                 | 2.107     | 15.209      | 200.757                 | 8.08        |
| 2000       | 13.70                 | 1.929     | 13.920      | 190.704                 | 9.80        |
| 2200       | 14.10                 | 1.750     | 12.631      | 178.098                 | 11.89       |
| 2400       | 14.30                 | 1.571     | 11.342      | 162.193                 | 14.44       |
| 2600       | 14.45                 | 1.393     | 10.053      | 145.270                 | 17.65       |
| 2800       | 14.52                 | 1.214     | 8.764       | 127.259                 | 21.80       |
| 3000       | 14.50                 | 1.036     | 7.476       | 108.395                 | 27.39       |
| 3200       | 14.40                 | 0.857     | 6.187       | 89.088                  | 35.30       |
| 3400       | 14.20                 | 0.679     | 4.898       | 69.548                  | 47.37       |
| 3600       | 13.80                 | 0.500     | 3.609       | 49.803                  | 68.07       |

# Calculations (Ratio and Torque)

Torque and Speed with low gearbox ratio

| Engine rpm | Torque output (lb-ft) | CVT ratio | Total ratio | Torque on wheel (lb-ft) | Speed (mph) |
|------------|-----------------------|-----------|-------------|-------------------------|-------------|
| 1800       | 13.20                 | 2.107     | 31.608      | 417.228                 | 3.89        |
| 2000       | 13.70                 | 1.929     | 28.929      | 396.334                 | 4.72        |
| 2200       | 14.10                 | 1.750     | 26.251      | 370.137                 | 5.72        |
| 2400       | 14.30                 | 1.571     | 23.572      | 337.082                 | 6.95        |
| 2600       | 14.45                 | 1.393     | 20.894      | 301.911                 | 8.49        |
| 2800       | 14.52                 | 1.214     | 18.215      | 264.480                 | 10.49       |
| 3000       | 14.50                 | 1.036     | 15.536      | 225.275                 | 13.18       |
| 3200       | 14.40                 | 0.857     | 12.858      | 185.149                 | 16.98       |
| 3400       | 14.20                 | 0.679     | 10.179      | 144.540                 | 22.79       |
| 3600       | 13.80                 | 0.500     | 7.500       | 103.503                 | 32.76       |



# Calculations (Shaft)

- Equation:

$$S_e = K_a \times K_b \times K_c \times K_d \times K_e \times K_f \times S_{ut}$$

$$D = \left( \frac{16 \times 2}{\pi} \right) \times \left( \sqrt{\frac{4 \times (K_f \times \text{Moment})^2}{(1000 \times S_e)}} \right) + \left( \sqrt{\frac{3 \times (K_{fs} \times \text{Torque})^2}{(1000 \times S_{ut})}} \right)$$

# Calculations (Shaft)

Factors:

$$Ka = a \times Sut^b = 0.9128$$

$$Kb = 0.879 \times d^{-0.107}$$

$$Kc = 0.59$$

$$Kd = \frac{ST}{SRT} = 1$$

$$Ke = 0.897$$

$$Kf = 1 + q \times (Kt - 1) = 1.42$$

$$qts = 0.8$$

$$q = 0.7$$

$$Kt = 1.6$$

$$Kts = 1.3$$

Moment = 3616.96 KN-mm

Torque = 565376 N-mm

Result:

$$D = 25.38\text{mm}$$

# Bearing selection

- Based on the shaft diameter  $D = 21.15\text{mm}$  our team chose the bearing that can fit the shaft diameter.

# Key selection

Table 7-6 from machine design book shows that

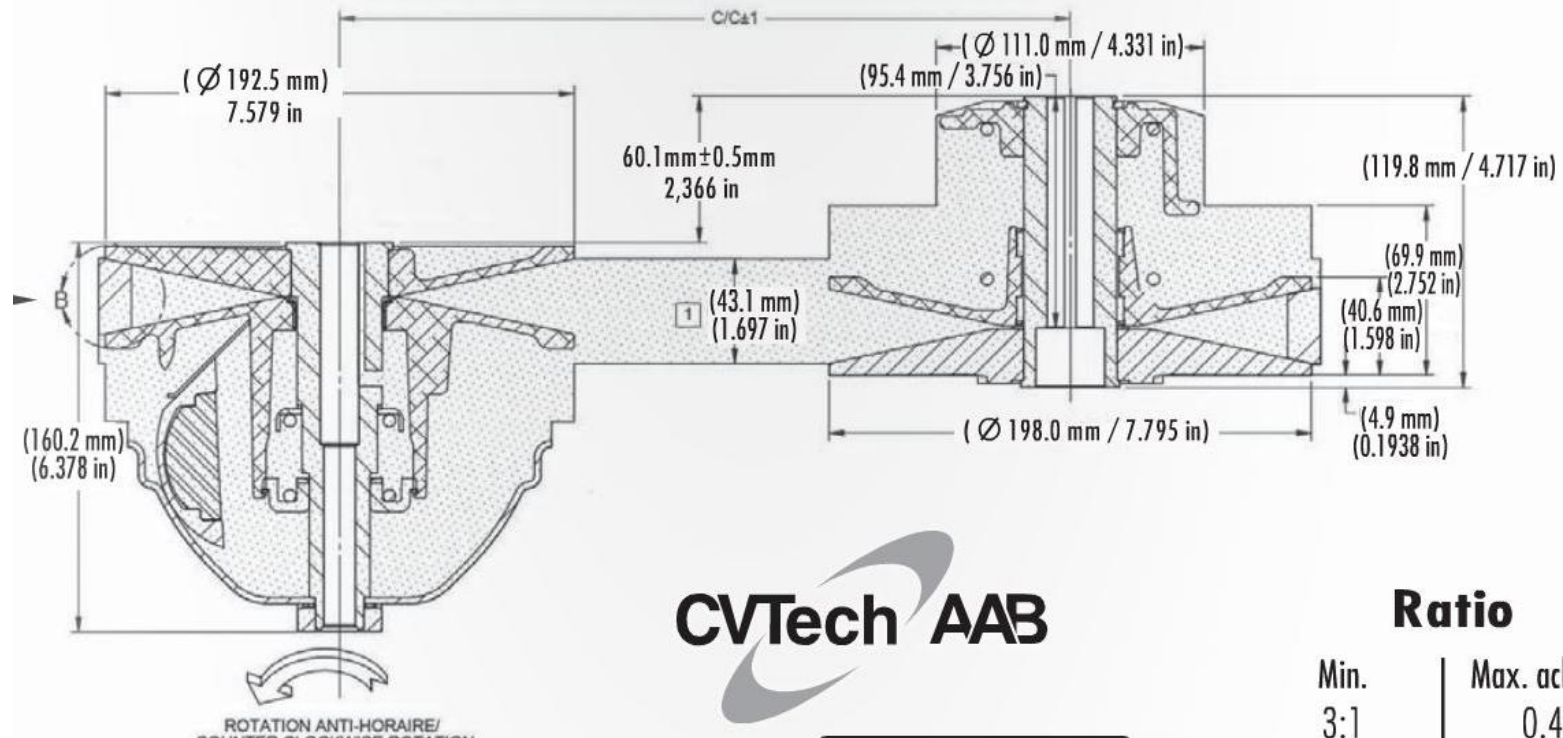
| Shaft diameter |                | Key Size |       | Keyway Depth |
|----------------|----------------|----------|-------|--------------|
| Over(in)       | To(incl.) (in) | W(in)    | H(in) | (in)         |
| 9/16           | 7/8            | 3/16     | 1/8   | 1/16         |

# Analysis (Results)

- CVT : 0.45 high speed ratio to 3.1 low speed ratio
- Gearbox: 2.734 high speed ratio, 5.682 low speed ratio
- Max torque on the wheel: 417.228 lb-ft
- Max speed: 68.07 mph
- Out put shaft diameter:  $D = 25.38\text{mm}$

# Chosen CVT

## PULLEY SERIES 0600 AND DRIVEN PULLEY SERIES 5600

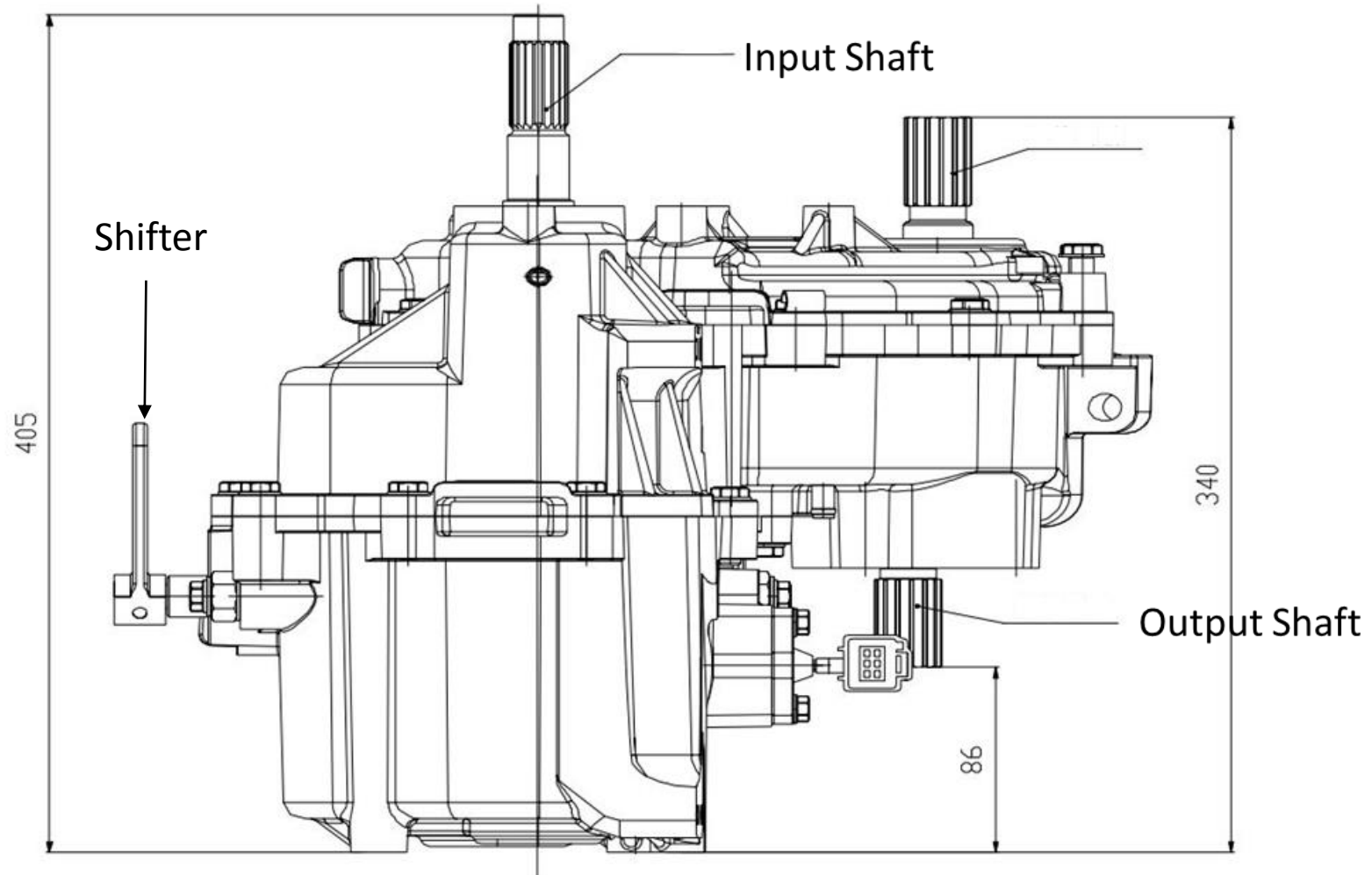


**CVTech AAB**

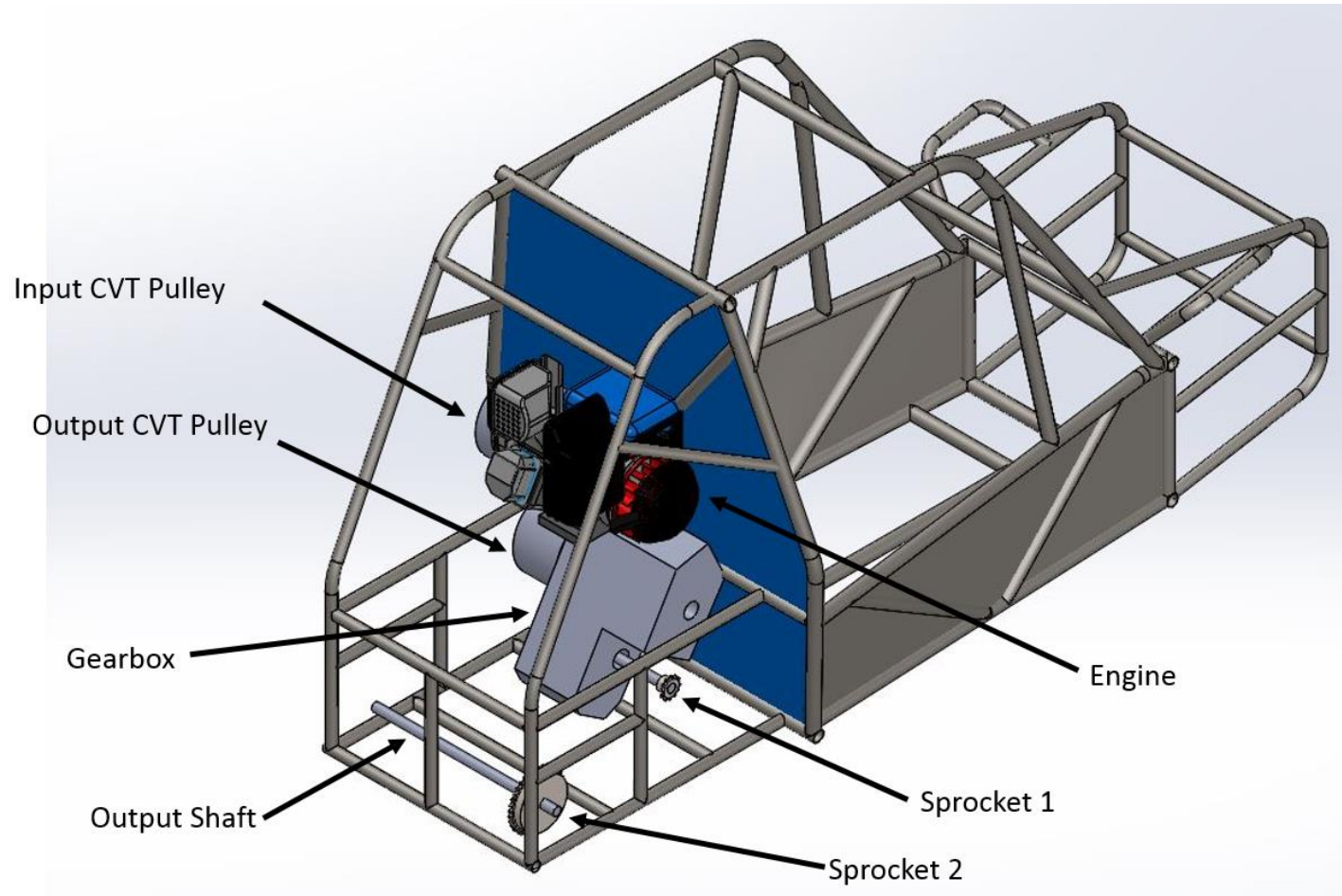
**Ratio**

| Min. | Max. achievable |
|------|-----------------|
| 3:1  | 0.45:1          |

# Chosen Gearbox

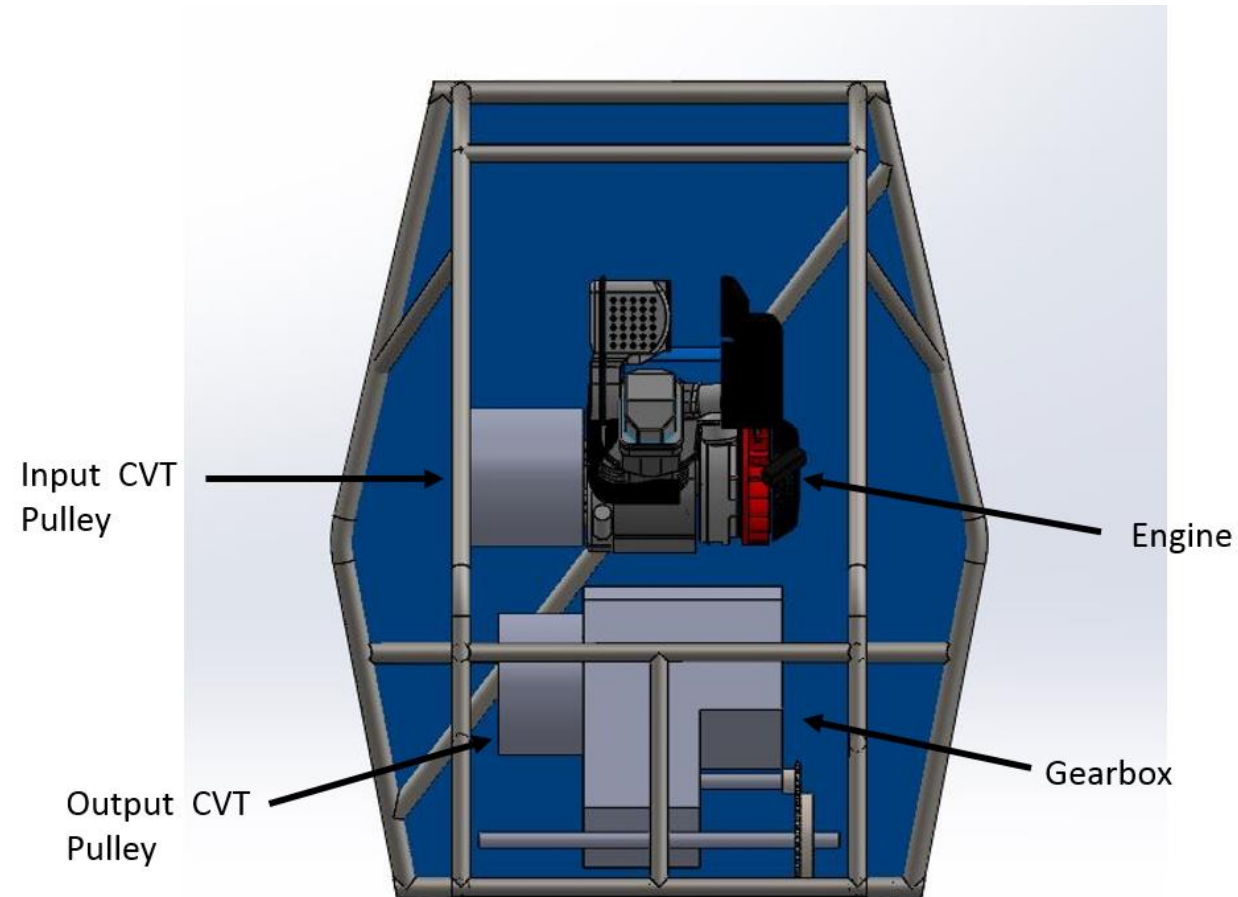


# Final Design (CAD)





# Final Design (CAD)



# Cost Analysis

- Budget
- Labor Fee Calculation
- Bill of Materials

# Cost Analysis

|                                      | Price(\$) | Quantity | Comments     | Total       |
|--------------------------------------|-----------|----------|--------------|-------------|
| Engine                               | 200       | 1        | Ship fee     | 200         |
| Gearbox                              | 400       | 1        | GaoKin       | 400         |
| CVT                                  | 580       | 1        | CV-Tech      | 580         |
| Shaft                                | 28.5      | 2        | Metals Depot | 57          |
| Bearing                              | 15        | 4        | Polaris      | 60          |
| Key                                  | 5         | 4        | Metal Depot  | 20          |
| Sprocket                             | 16        | 4        | G & G        | 64          |
| Chain                                | 15        | 2        | G & G        | 30          |
| Half-shaft                           | 260       | 4        | Polaris      | 1040        |
| Shipping                             | 200       |          |              | 200         |
| <b>Total Price<br/>(include tax)</b> |           |          |              | <b>2451</b> |

# Labor Fee Calculation

- Units per week: 80 units
- Complete units per day: 16 units
- Time per unit job: 5 hours
- Hours of work per day: 80 hours
- Number of laborers: 10 people
- Hourly wage: \$26

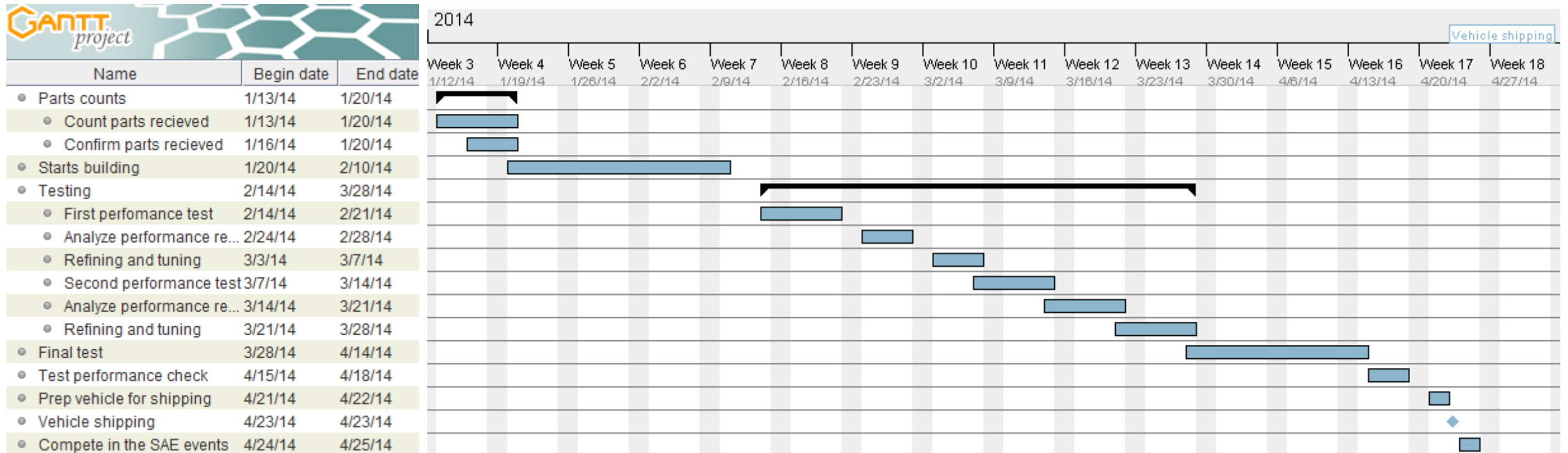
| Total work units | Complete units per day | Total work hours per person | Number of laborers | Work hours per person per day | Hourly Wage(\$) | Total cost of labors |
|------------------|------------------------|-----------------------------|--------------------|-------------------------------|-----------------|----------------------|
| 4000             | 15                     | 2086                        | 10                 | 8                             | 26              | 542,286              |

# Bill of Materials

- These prices are based upon whole sale costs or approximately 50% of listed price.

| 12 tooth sprockets | 36 tooth sprockets | Half shafts | 2 feet 1040 Steel Shaft | Engine  | CVT       | Gearbox | Chain | Total for single unit | Total     |
|--------------------|--------------------|-------------|-------------------------|---------|-----------|---------|-------|-----------------------|-----------|
| 10,120             | 30,360             | 526,000     | 5,700                   | 979,980 | 1,160,000 | 800,000 | 5836  | 879                   | 3,511,996 |

# Project plan- Spring 2014



# Conclusion

- Final design: CVT, 4 step gear box, and 3:1 sprocket reduction.
- Our drive-train will be successful in the SAE Baja Competition.
- This drive-train has theoretically satisfied all customer needs.
- We have designed it to fit the parameters of the frame team.
- Our team will order parts on schedule and commence building.

# References

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# References

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- Continuously variable transmission(CVT)  
[https://d2t1xqejof9utc.cloudfront.net/files/19153/eti\\_19\\_CVTransmission.pdf?1363999370](https://d2t1xqejof9utc.cloudfront.net/files/19153/eti_19_CVTransmission.pdf?1363999370)
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# References

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- Roller Chain 4 Less

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